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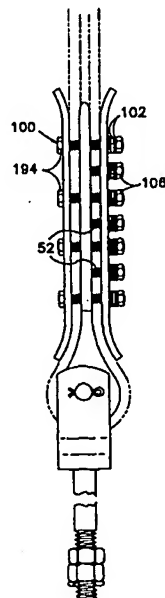
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(54) Title: FLAT CABLE CONNECTING DEVICE

(57) Abstract

A tension member termination device optimized for terminating flexible flat tension members, the device including a socket with a compression portion and a bulbous portion and a compression plate on each side of the compression portion fastenable by fasteners extending through all of these features. The device provides a pathway for the tension member through the device and upon torquing the fasteners reliably secure the tension member while avoiding deleterious pressure and stress therein. The invention also provides a safety clamp for optional use with the invention.



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CABLE CONNECTING DEVICE

Technical Field

The present invention relates to elevator systems. More particularly the invention relates to a termination for a flexible flat tension member.

5 Background of the Invention

A conventional traction elevator system includes a car, a counterweight, two or more ropes (tension members) interconnecting the car and counterweights; terminations for each end of the ropes at the connection points with the car and counterweights, a traction sheave to move the ropes and a machine to rotate the
10 traction sheave. The ropes have traditionally been formed of laid or twisted steel wire which are easily and reliably terminated by means such as compression terminations and potted terminations.

Compression type terminations for steel ropes of larger diameters (conventional steel elevator ropes) are extremely effective and reliable. The range
15 of pressures placed on such terminations is reasonably broad without adverse consequence. Providing that the pressure applied is somewhere reasonably above the threshold pressure for retaining the ropes, the termination is effective.

With an industry trend toward flat ropes, those ropes having small cross-section cords and polymeric jackets, significantly more criticality is involved in
20 effectively terminating the same. More specifically, the polymeric coating can creep to even 50% of its original thickness when subjected to pressure. Prior art knowledge which teaches one to exceed a threshold is not all that is of concern for flexible flat tension members. Upper limits on compression are also important.

Since current knowledge in the art of tension member terminations is less
25 than sublime for flexible flat tension members due both to the small cord diameter and the jacket properties discussed above, the art is in need of a tension member terminating device which specifically optimizes terminations of the flexible flat

tension members currently emerging in the field.

Disclosure of the Invention

5 According to the present invention, a compression termination device having a friction enhancing construction while reducing compressive forces applied to the tension member, comprises a load side plate, a cut side plate and a socket, a portion of which being receivable between said load side and cut side plates and a portion of which is bulb shaped. The plates and socket are of
10 sufficient width to accept a flexible flat tension member of a selected width and are securable together by fasteners. In a condition where the fasteners are loose, the tension member is insertable between the load side plate and socket, toward and around the bulb and back up between the cut side plate and socket whereafter tightening of the fasteners produces significant frictional forces on the tension
15 member to retain the same while compressive forces on the tension member are intentionally limited to about 2 Mpa on the load side of the device and 5 Mpa on the cut side of the device. Friction is increased by texturing the surfaces of the termination device with which the tension member makes contact. With
20 compressive forces as stated, creep is minimized while the termination maintains a sufficient gripping force to provide a factor of safety (fos) of 12 to maintain adequate strength of the termination.

 Since creep is a possibility even with Mpa levels at the stated limits, the invention optionally includes a structure providing resilience such that compressive force on the tension member will remain in the acceptable range even
25 if creep does occur.

 The termination of the invention further optionally includes a jamming device attachable to the cut end of the tension member. In the unlikely event of tension member slippage through the termination device, the jamming device will be drawn into the termination device and will prevent the tension member cut end
30 from pulling through the termination device.

Brief Description of the Drawings

FIGURE 1 is a perspective view of an elevator system;

FIGURE 2 is an exploded perspective view of the termination device of
5 the invention;

FIGURE 3 is an end elevation view of a socket of the invention;

FIGURE 4 is a side elevation view of a socket of the invention;

FIGURE 5 is a top plan elevation view of a socket of the invention;

FIGURE 6 is a view similar to FIGURE 3 but having studs installed
10 therein;

FIGURE 7 is an end elevation view of a compression plate of the
invention;

FIGURE 8 is a side elevation view of a compression plate of the invention;

FIGURE 9 is an end elevation view of the invention in an assembled and
15 torqued condition;

FIGURE 10 is a side elevation view of the invention in an assembled and
torqued condition;

FIGURE 11 is a schematic view of a nut and bolt with belleville washers
thereon in the uncompressed condition;

FIGURE 12 is a schematic view of a nut and bolt with belleville washers
20 thereon in the compressed condition;

FIGURE 13 is a schematic view of an alternate biasing means of the
invention;

FIGURE 14 is a schematic view of the termination device of the invention
25 illustrating force directions for calculations provided herein;

FIGURE 15 is a perspective view of the pivot connector of the termination
device of the invention;

FIGURE 16 is a perspective assembled view of the jamming device of the
invention;

30 FIGURE 17 is a perspective view of the interior portion of one side of the

jamming device; and

FIGURE 18 is a perspective view of the interior portion of a second side of the jamming device.

FIGURE 19 is a sectional, side view of a traction sheave and a plurality of flat ropes, each having a plurality of cords.

FIGURE 20 is a sectional view of one of the flat ropes.

Best mode for carrying out the Invention

Referring to FIGURE 1, the relative location of the tension member termination device of the invention can be ascertained. For clarity, the elevator system 12 is illustrated having car 14, counterweight 16, a traction drive 18 and a machine 20. The traction drive 18 includes a tension member 22 interconnecting car 14 and counterweight 16 which member is driven by sleeve 24. Both ends of tension member 22 i.e., car end 26 and counterweight end 28 must be terminated. It is this termination point for a flexible flat tension member with which the invention is concerned. An exemplary tension member of the type contemplated in this application is discussed in further detail in U.S. Serial No. 09/031,108 filed February 26, 1998 Entitled Tension Member For An Elevator and Continuation-In-Part Application Entitled Tension Member For An Elevator filed December 22, 1998 under Attorney Docket No. 98-2143, both of which are entirely incorporated herein by reference. The elevator system depicted is provided for exemplary purposes to illustrate the location of the device of the invention.

Focusing on the termination device, referring to FIGURE 2, and noting that both ends 26 and 28 may be similarly terminated, the device of the invention comprises, principally, a socket 30 around which a flat flexible tension member extends (not shown), a load side plate 80 and a cut side plate 96. The invention further comprises a resilient compression subsystem and a pivoted connector which will be discussed hereinbelow.

Returning to the principal portion of the invention and directing attention to FIGURES 2-5, socket 30 includes a tapered end 32 to both ease insertion of a tension member in the loosely assembled condition of the device and additionally and importantly to avoid a sharp edge which would otherwise promote fatigue in the tension member where the member enters the termination device 10. The taper is from both major surfaces of socket 30 i.e., load surface 34 and cut surface 36. Socket 30 further includes troughs 38 and 40, respectively. Troughs 38 and 40 are sized to receive a tension member of a width that has been pre-selected. Each trough nests with a section of the tension member when the termination device is assembled. Each trough may be left smooth and the termination device will remain effective. It is preferred, however, to texture each trough and the bulb surface 42 thereby increasing the coefficient of friction of all surfaces of socket 30 with which the terminated tension member will make contact. A preferred method for texturing troughs 38 and 40 as well as surface 42 is by sand blasting. It will be understood however that other methods such as machining, chemical etching, etc. could also be used.

Socket 30 further includes binding wings 44 and 46 having a plurality of fastener clearance holes 48 and, in a preferred arrangement, a plurality of stud receiving openings 50. The number of holes 48 depends upon the length socket 30 and the allowable pressure on the tension member. In the embodiment of FIGURES 3 and 4, four holes 48, and three openings 50 are provided on each wing 44 and 46. In a preferred embodiment, openings 50 are threaded to receive studs 52 (FIGURE 6). It should be noted that studs 52, as shown in FIGURE 6 extend only toward the cut side 36 of socket 30. Studs 52 enable the application of a greater compressive load on cut side 36 of socket 30 than the load applied on load side 34 of socket 30 which is applied by bolts extending completely through device 10. In other words, the load placed on the respective sides of socket 30 (through plates discussed hereunder) by the bolts (which extend through the device) and nuts is approximately equal; studs 52 allow more load to be placed on the cut side as is desirable and explained further hereinafter.

In a preferred embodiment, socket 30 (the section bound between the plates) is about 9 to about 12 millimeters thick to support the stress placed thereon.

Referring back to FIGURE 5, surface 42 is illustrated as a depressed area
5 between shoulders 54 and 56. The shoulders are preferably provided to assist in properly seating a tension member when the termination is being constructed. This helps to ensure that the load bearing cords of the tension member do not experience significantly unequal loading. Significant shoulder height is not necessary to achieve the desired result. A height of about 1 millimeter for each
10 shoulder has been found to function adequately.

The final feature of socket 30 is pin receptacle 58 which preferably includes bushing 60 therein. Pin receptacle 58 is located in bulb 62 of socket 30 but is offset from the center axis of bulb 62. More specifically, and to minimize angular stress in the tension member, receptacle 58 is offset toward the load side
15 34 of socket 30 and is positioned to be aligned on center with a tension member assembled with said termination member. By so locating the receptacle, and thus the pivot point in the system, the load hanging therefrom is aligned with the load side of the tension member engaged with the termination device of the invention.

Socket 30 is important to the functionality of the termination device of the
20 invention principally because it provides three distinct friction zones and a smooth bend surface for the tension member. The combination reduces the compression force required to prevent tension member slippage which is particularly helpful where flexible flat tension members having polymeric jackets are employed. Reducing the compression force that would otherwise be required, alleviates creep
25 and reduces stress in the tension member. This is desirable since it may reduce the number of re-roping operations that would be carried out during the life of the elevator.

Thus far only the socket 30 has been described and it will be apparent to one of ordinary skill in the art that the socket alone does not retain the tension
30 member. Reference is, therefore, made to FIGURES 7 and 8 where the load side

and cut side plates 80 and 96, respectively, are described. It should be noted that plate 80 and plate 96 are identical in a preferred embodiment and are provided distinct numerals merely to distinguish each side of the termination device (which is side dependent) rather than to signify any distinction between the plates themselves.

Plates 80 and 96 are curved at longitudinal top 82 and bottom 84 ends thereof. The degree of the curvature is selected to, at end 82, reduce fatigue of the tension member at the point where it enters the termination device. The curve at 82 preferably mirrors the tapered end 32 of socket 30. Bottom end 84 is curved to match the transition from the compression portion of socket 30 to bulb 62. In a preferred embodiment, the curves at 82 and 84 as well as those in the opposite plate 96 are identical so that plates 80 and 96 are interchangeable and orientable in either direction. This facilitates assembly of the termination device.

On the convex side 86 of each plate 80 and 96 (it should be noted that the sub numerals employed to describe features of each plate will be identical because the features are identical and no distinction as to side of the termination device is necessary), a region 88 is provided where a textured surface is desirable. The texture may be of any type that increases the coefficient of friction without being significantly deleterious to the jacket of the tension member. In one preferred embodiment sand blasting of the region is indicated. It will be understood that the region may be textured by machining, chemical etching, knurling, etc. if desired or otherwise indicated. A preferred range of friction for the device of the invention is about .15 to about .5. Region 88 is outlined in FIGURE 8 in phantom lines.

Due to the texturing processes, and especially the sand blasting process, the termination device may become more susceptible to corrosion. In order to avoid or inhibit such corrosion, it has been determined that yellow zinc plating may be advantageously used. Alternatively, stainless steel material or aluminum material may be used for the device of the invention.

Bordering Region 88 on each longitudinal side thereof are a plurality of clearance holes 90. In a preferred embodiment, seven holes 90 are provided on

- each side of Region 88. Holes 90 accept through passage of bolts to assemble device 10 and also studs 52 discussed with reference to FIGURE 6. Although it has been stated that plates 80 and 96 are preferably interchangeable, it is possible to eliminate holes on the load side plate 80 which correspond to studs 52
- 5 estimating only from the cut side 36 of socket 30. The holes that can be eliminated may be ascertained by reference to FIGURE 9 wherein bolts 100 are illustrated as extending through the entire assembly and studs 52 only extend through one side thereof, therefore only requiring clearance holes 90 in the cut side plate.
- 10 Referring to FIGURES 9 and 10, the device 10 is illustrated in the assembled condition with bolts 100 and studs 52 properly torqued. The torque applied is discussed further hereunder but is dictated by the allowed pressure on the tension member which is about 2 Mpa on the load side and about 5 Mpa on the cut side of the termination device 10.
- 15 Preferably a biasing arrangement is included in the assembly of device 10, more specifically, it is desirable to anticipate possible creep of the tension member and therefore provide means to maintain the prescribed normal force on the tension member even if it is reduced in thickness by the effects of creep. One such arrangement is illustrated in FIGURES 11 and 12. In FIGURE 11, the biasing
- 20 arrangement of a stack of belleville washers 102 is illustrated in the uncompressed state. FIGURE 12 on the other hand, illustrates the same stack of washers 102 after torquing of the bolt 100. In the event the volume of material bound between a bolt head 194 and nut 106 (FIGURE 9) decreases after torquing, due to creep of the tension member, washers 102 will expand and maintain the pressure on the
- 25 tension member. The normal pressure on the tension member will thus be maintained. The additional benefit of easy visual inspection for creep is realized by the invention since if the washers exhibit a spaced appearance like that of FIGURE 11, retorquing is required. Belleville washers are known to the art and do not require specific explanation. Other biasing means are also employable with
- 30 the device of the invention with the joining concept being that the predetermined

normal force on the tension member be maintained. One alternate biasing means is a corrugated spring metal sheet 100 which would be placed atop cut side plate 96 in place of washers 102. Sheet 110 has holes 112 for through passage of bolts 100 or studs 52 depending upon location. Holes 112 are preferably slotted to
5 allow for longitudinal expansion of the spring sheet during torquing of fasteners and consequent compression of spring sheet 110.

Referring now to FIGURE 14, a schematic view of the invention with the plates exploded from the socket and with the forces and tensions required indicated. The invention provides five friction areas which combine to form three
10 friction zones. The areas include: (1) the inside surface of the load side plate which contacts one side of the tension member; (2) the load side of the socket (corresponds to load plate) providing friction on an opposite side of the tension member from the load side plate; (3) the bulbous section which provides a continuous frictional surface on which the tension member is in contact; (4) the
15 cut side of the socket and (5) the cut side plate inside surface, surfaces 4 and 5 being opposed. These five areas create three friction zones that are resolved in the following equations to determine adequacy of the assembly. Each zone is mathematically quantifiable. The sum of the three frictions must be sufficient to prevent slippage. Practically speaking, it is desirable to attain a 100% holding
20 efficiency. In order to achieve this efficiency, the sum of the three friction zones must be equal to or exceed the breaking strength of the tension member being employed. With an assembly having a 100% holding efficiency, the tension member will break before the termination device allows the tension member to slip. In the following equations, several assumptions are made: The rope
25 breaking strength is 30,000 Newtons; the coefficient of friction (μ) for the sand blasted surfaces that are preferred in the invention is .25; and the plate normal force is a function of the number of bolts employed multiplied by 1540 Newtons which is the expected force delivered by each bolt. These numbers are exemplary and clearly can be adjusted depending upon circumstances. One of ordinary skill
30 in the art following exposure to this disclosure should be fully capable of adjusting

the calculations to conform to any specific parameters given without undue experimentation. FIGURE 14 is informative and used in connection with the following formulas employed to determine gripping strength of device 10 and stress in various components.

5

SUPPOSE HITCH TENSION IS DIVIDED INTO 3 REGIONS:

$$T_1 \rightarrow T_2 \quad (\text{Region 1})$$

$$5 \quad T_2 - T_3, \quad (\text{Region 2})$$

$$\text{and} \quad T_3 - T_4 \quad (\text{Region 3})$$

we know, T_1 = flexible flat tension member breaking strength

$$10 \quad \text{and } T_4 = 0,$$

since if $T_4 > 0$ tension member will slip in the termination device

FOR EXAMPLE, ASSUME

Region 1:

15

$$T_1 = 30,000 \text{ N} = \text{tension member Breaking Strength}$$

$$\mu = .25 = \text{coefficient of friction}$$

$$N_1 = \text{Plate normal force}$$

$$= 12,320 \text{ N} \quad (8 \text{ bolts} \times 1540 \text{ N})$$

20

for region 1 (referring to Figure 14) $F_1 = \mu N_1$

$$F_1 = \mu (N_1) \text{ 2 plates}$$

$$F_1 = .25 (12,320) \text{ 2 plates}$$

$$F_1 = 6160 \text{ N}$$

25

$$\text{and} \quad T_2 = T_1 - F_1$$

$$\text{so} \quad T_2 = (30,000 - 6160) =$$

$$30 \quad = 23,840 \text{ N}$$

Region 2: FROM TRACTION THEORY WE KNOW:

$$\frac{T_2}{T_3} = e^{\mu\theta} \text{ or } T_3 = \frac{T_2}{e^{\mu\theta}}$$

5

$$T_3 = \frac{23,840}{e^{(.25)(\pi)}} = \frac{23,840}{2.291}$$

$$T_3 = 10,405 \text{ N}$$

10

Region 3:

15

From Previous Calculations,

$$T_3 = 10,405 \text{ N}$$

and T_4 must be ≤ 0 (values greater than 0 indicate tension member slippage)

20 Cut side plate has 14 fasteners x 1540N (the studs 52 are available only to the cut side plate)

Assume $N_2 > N_1 = 21,560 \text{ N}$, and then calculate for slippage

$$T_4 = T_3 - F_2$$

25 and $F_2 = \mu (N_2) \text{ 2 plates}$

$$F_2 = .25 (21,560) 2$$

$$F_2 = 10,780 \text{ N}$$

30

CRITERIA:

IF $F_2 \geq T_3$,

design is adequate, tension member will not slip

5 $F_2 > T_3$? (YES)

10,780N > 10,405N, so

design is adequate

10 PRESSURE ON URETHANE tension member:

Example I:

125 mm long

Tension member is 30 mm wide

Pressure = $\frac{N}{A} = \frac{11000 \text{ N}}{30 \text{ mm} \cdot 125 \text{ mm}}$

15 A 30 mm . 125 mm

= 2.933 MPa = 425 psi

In this example the pressure is beyond that taught in the invention

20

Example II

Tension member plates are 190 mm long
30 mm wide

5

LOAD SIDE

$$\text{Pressure} = \frac{N}{A} = \frac{12320N}{30.190} = 2.16 \text{ MPa} = \underline{313 \text{ psi}} \quad (\text{LOAD})$$

10

CUT SIDE

$$\frac{21560N}{30.190}$$

$$= 3.78 \text{ MPa} \quad \underline{548 \text{ psi}} \quad (\text{CUT})$$

$$30.190$$

IN THIS EXAMPLE THE PRESSURE EXERTED ON THE TENSION
15 MEMBER IS ACCEPTABLE FOR BOTH SIDES OF THE TERMINATION
DEVICE. THUS, PLATES ARE LONG ENOUGH.

BOLT TORQUE CALCULATIONS (for first example only):Example I - 125 mm plates with 8 bolts.

LOAD PER BOLT

$$5 \quad N_1 = N_2 = 11,000 \text{ N}$$

$$\text{LOAD PER BOLT} = \frac{11,000}{8} = 1375 \text{ N}$$

BOLT SIZE/THREADS:

10 M8 - 8 mm course thread Pitch = 1.25

PROP CLASS 8.8

BOSSARD CATALOG TABLE, PRELOAD TORQUE

	<u>PRELOAD</u>	<u>TORQUE</u>	
15	17,050 N	24 N-M	BOSSARD CATALOG
	So for 1,540 N	$\frac{1540}{17,050} (24) = 2.17$	N-M

$$T = 0.2 F_t d$$

$$20 \quad = 0.2 (1540) 8 = 2.5 \text{ N-M}$$

where $F_t = 1540 \text{ N}$ and $d = 8 \text{ mm}$

PLATE DIMENSIONAL CALCULATIONS

$$I^{3/16} \text{ PLATE} = \frac{1 \text{ (}^3/16\text{)}^3}{12} = .0005493 \text{ ''}$$

$$5.4931 \times 10^{-4}$$

5

$$\Delta = \frac{5 w l^4}{384 E I}$$

$$I_{\frac{1}{16}} = \frac{1 \text{ (.25)}^3}{12} = .001302$$

10

$$\Delta = \frac{5 (425) (1.653)^4}{384 (3 \times 10^7) (.0005493)} = \frac{1.302 \times 10^{-3} \text{ in}^4}{}$$

$$15 \quad \Delta = .002507 \text{ in}$$

$$\text{if } \Delta = \frac{P \ell^3}{48 E I} = \frac{425 (1.181) \text{ } 1.653^3}{48 (5.493 \times 10^{-3}) (3 \times 10^7)}$$

20

$$= .002866 \text{ in (}^3/16\text{)}$$

$$\sim = \frac{M_c}{I} = \frac{[145.159] \text{ } .1875}{.0005493} = \frac{13.608}{.0005493}$$

Uniform Dist. Load

25

$$24,774 \text{ psi}$$

30

$$M_{\max} = \frac{w \ell^2}{8} = \frac{425 (1.653)^2}{8} = 145.159$$

UNIFORM DIST. LOAD.

$$5 \quad \sigma = \frac{M_c}{I} = \frac{[145.159] [2.50]}{1.302 \times 10^{-3}} = 13,935 \text{ psi}$$

Referring to FIGURE 15, a clevis is illustrated. Clevis 120 is seen connected to the termination assembly in FIGURE 2 (in exploded condition). The clevis is conventional and will be easily recognized by one of skill in the art. The clevis 120 is employed to provide a pivot point near a terminal end of the loaded tension member to reduce vibratory fatigue therein. Clevis 120 is connected to socket 30 by pin 122 extending through receptacle 58.

Referring now to FIGURES 16-18, an optional device 130 for use with the termination device 10 is illustrated. The purpose of device 130 is to jam with termination device 10 in the unlikely event of tension member slippage through device 10. Device 130 is clamped onto the cut end of the tension member somewhere beyond region T4 as discussed above. When engaged with the tension member, device 130 cannot move thereon. Thus, if the tension member slipped it would draw device 130 into contact with cut side plate 96 and side 36 of socket 30 and would jam there preventing further slippage.

Device 130 comprises a female portion 132 (FIGURE 17) and a male portion 150 (FIGURE 18). Female portion 132 features a tension member groove 134 approximately the thickness of the tension member which is intersected by crimp grooves 136 and 138. Bore holes 140 are provided for through passage of fasteners 142. Male portion 150 provides tension member deformation ridges 152 and 154 which are intended to extend into grooves 136 and 138, respectively upon assembly of device 138. Portion 150 further includes holes 156 which are coaxially with holes 140 when device 130 is assembled to facilitate through passage of assembly bolts 142.

In use, a cut end of a tension member, i.e., the end not being used to support the elevator, is inserted in groove 134 and portion 150 is placed in position. When the bolts 142 are tightened, ridges 152 and 154 force the tension member to follow a tortuous path around the ridges and into grooves 136 and 138. In this way the tension member is prevented from moving relative to device 130 and if device 130 moves into contact with device 10 to tension member slippage, the slippage will be arrested.

A principal feature of the present invention is the flatness of the ropes used in the above described elevator system. The increase in aspect ratio results in a rope that has an engagement surface, defined by the width dimension "w", that is optimized to distribute the rope pressure. Therefore, the maximum rope pressure is minimized within the rope. In addition, by increasing the aspect ratio relative to a round rope, which has an aspect ratio equal to one, the thickness "t1" of the flat rope (see FIGURE 19) may be reduced while maintaining a constant cross-sectional area of the portions of the rope supporting the tension load in the rope.

As shown in FIGURE 19 and 20, the flat ropes 722 include a plurality of individual load carrying cords 726 encased within a common layer of coating 728. The coating layer 728 separates the individual cords 726 and defines an engagement surface 730 for engaging the traction sheave 724. The load carrying cords 726 may be formed from a high-strength, lightweight non-metallic material, such as aramid fibers, or may be formed from a metallic material, such as thin, high-carbon steel fibers. It is desirable to maintain the thickness "d" of the cords 726 as small as possible in order to maximize the flexibility and minimize the stress in the cords 726. In addition, for cords formed from steel fibers, the fiber diameters should be less than .25 millimeters in diameter and preferably in the range of about .10 millimeters to .20 millimeters in diameter. Steel fibers having such diameter improve the flexibility of the cords and the rope. By incorporating cords having the weight, strength, durability and, in particular, the flexibility characteristics of such materials into the flat ropes, the traction sheave diameter "D" may be reduced while maintaining the maximum rope pressure within acceptable limits.

The engagement surface 730 is in contact with a corresponding surface 750 of the traction sheave 724. The coating layer 728 is formed from a polyurethane material, preferably a thermoplastic urethane, that is extruded onto and through the plurality of cords 726 in such a manner that each of the individual cords 726 is restrained against longitudinal movement relative to the other cords 726. Other materials may also be used for the coating layer if they are sufficient to meet the

required functions of the coating layer: traction, wear, transmission of traction loads to the cords and resistance to environmental factors. It should be understood that although other materials may be used for the coating layer, if they do not meet or exceed the mechanical properties of a thermoplastic urethane, then the benefits resulting from the use of flat ropes may be reduced. With the thermoplastic urethane mechanical properties the traction sheave 724 diameter is reducible to 100 millimeters or less.

As a result of the configuration of the flat rope 722, the rope pressure may be distributed more uniformly throughout the rope 722. Because of the incorporation of a plurality of small cords 726 into the flat rope elastomer coating layer 728, the pressure on each cord 726 is significantly diminished over prior art ropes. Cord pressure is decreased at least as $n^{-1/4}$, with n being the number of parallel cords in the flat rope, for a given load and wire cross section. Therefore, the maximum rope pressure in the flat rope is significantly reduced as compared to a conventionally roped elevator having a similar load carrying capacity. Furthermore, the effective rope diameter 'd' (measured in the bending direction) is reduced for the equivalent load bearing capacity and smaller values for the sheave diameter 'D' may be attained without a reduction in the D/d ratio. In addition, minimizing the diameter D of the sheave permits the use of less costly, more compact, high speed motors as the drive machine.

A traction sheave 724 having a traction surface 750 configured to receive the flat rope 722 is also shown in FIGURE 19. The engagement surface 750 is complementarily shaped to provide traction and to guide the engagement between the flat ropes 722 and the sheave 724. The traction sheave 724 includes a pair of rims 744 disposed on opposite sides of the sheave 724 and one or more dividers 745 disposed between adjacent flat ropes. The traction sheave 724 also includes liners 742 received within the spaces between the rims 744 and dividers 745. The liners 742 define the engagement surface 750 such that there are lateral gaps 754 between the sides of the flat ropes 722 and the liners 742. The pair of rims 744 and dividers, in conjunction with the liners, perform the function of guiding the

flat ropes 722 to prevent gross alignment problems in the event of slack rope conditions, etc. Although shown as including liners, it should be noted that a traction sheave without liners may be used.

While preferred embodiments have been shown and described, various
5 modifications and substitutions may be made thereto without departing from the spirit and scope of the invention. Accordingly, it is to be understood that the present invention has been described by way of illustration and not limitation.

What is claimed is:

1. A tension member termination device for an elevator system comprising:
a socket having a bulbous end, said socket defining a tension member path therearound;
a load side plate affixable to said socket to apply a normal pressure to a
5 load side of an end of a tension member between said socket and said load side plate;
a cut side plate affixable to said socket to apply a normal pressure to a cut side of said end of said tension member between said socket and said cut side plate.
2. A tension member termination device for an elevator system as claimed in claim 1 wherein said path defined by said socket includes a surface which is textured to increase the coefficient of friction thereof.
3. A tension member termination device for an elevator system as claimed in claim 2 wherein said surface is sand blasted.
4. A tension member termination device for an elevator system as claimed in claim 1 wherein said load side plate and said cut side plates are affixed to said socket by a plurality of fasteners common to both plates.
5. A tension member termination device for an elevator system as claimed in claim 1 wherein said socket further includes studs extending from said socket in a direction to intersect said cut side plate enabling a greater compressive load to be placed upon said cut side plate than said load side plate.

6. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device comprising:
- 5 a socket having a relatively narrow section and a relatively bulbous section, said socket defining a tension member pathway therearound and a fastener section for through passage of fasteners;
- a load side plate fastenable to said socket by said fasteners, said load side plate extending along said relatively narrow section of said socket on a load side
- 10 thereof;
- a cut side plate fastenable to said socket by said fasteners, said cut side plate extending along said relatively narrow section of said socket on a cut side thereof.
7. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 6 wherein said
- 5 flexible flat tension member is located between said load side plate and said socket, extends around said bulbous section in contact therewith and between said cut side plate and said socket.
8. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 6 wherein said
- 5 socket further includes studs extending from said relatively narrow section and toward said cut side plate to provide additional compressive capability to said cut side plate.

9. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 8 wherein said
5 load side plate compresses said tension member to about 2 MPa.
10. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 8 wherein said cut
5 side plate compresses said tension member to about 5 MPa.
11. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 6 wherein said
5 socket and said load side plate and said cut side plate all include a textured surface corresponding to surfaces contacted by said tension member.
12. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 11 wherein said
5 surfaces are sand blasted.
13. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 6 wherein said
5 socket further includes a pivot pin receiver in said bulbous section.

14. An elevator system having an elevator car, a machine a counterweight and a flexible flat tension member extending between said counterweight and said elevator car, said tension member being terminated to at least one of said car and said counterweight by a termination device as claimed in claim 13 wherein said receiver is located in said bulbous section so as to be aligned with a load side of said tension member when engaged by said termination device.
15. A tension member termination device comprising:
a tension member compressor having at least one area that will contact a tension member having a traction enhanced surface thereon.
16. A tension member termination device comprising:
a compression member against which a tension member is compressible;
a load plane defined by said tension member in said compression member;
a pivot associated with said compression member, said pivot having a center which is aligned with said load plane.
17. A termination device for a tension member comprising:
a compressive system to engage and compress a tension member;
a biaser to maintain a selected compressive force on said tension member.
18. A spring loaded clamp for termination of a tension member comprising:
a clamp having at least two members;
a plurality of fasteners to fasten said members together in compressive relationship;
at least one biaser arranged to maintain said compressive relationship.

19. A method for terminating a tension member comprising:
feeding said tension member into a first side of a compressive system in one direction and looping said tension member back through a second side of said compressive system;
- 5 spring loading said compressive system;
compressing said compressive system around said tension member whereby said spring loading maintains a set compressive load on said tension member.
20. A method according to claim 19 wherein said compressing comprises compressing said first side of said compressive system to a first force and compressing said second side of said compressive system to a second force.
21. A method according to claim 20 wherein said second force is higher than said first force.

FIG.1

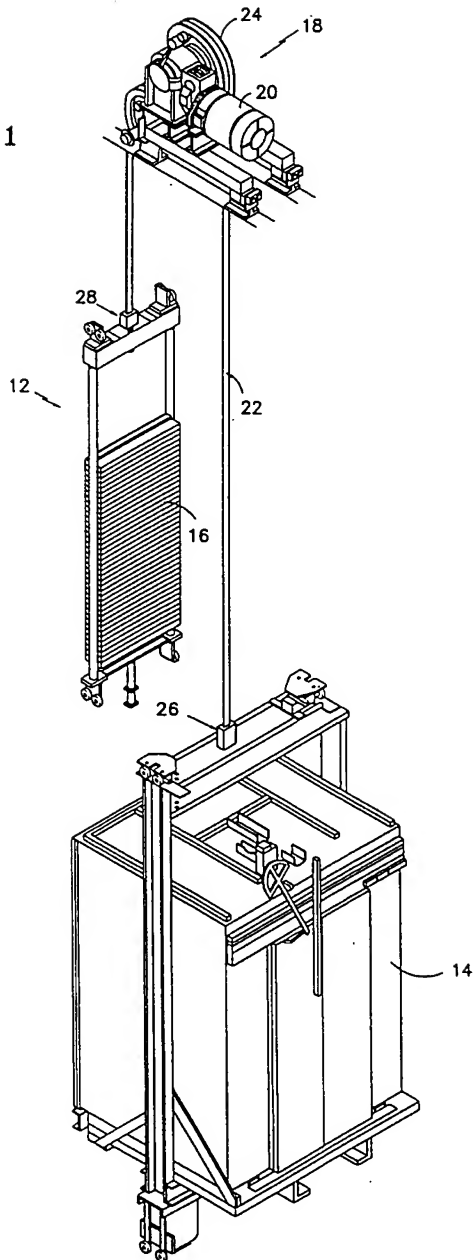
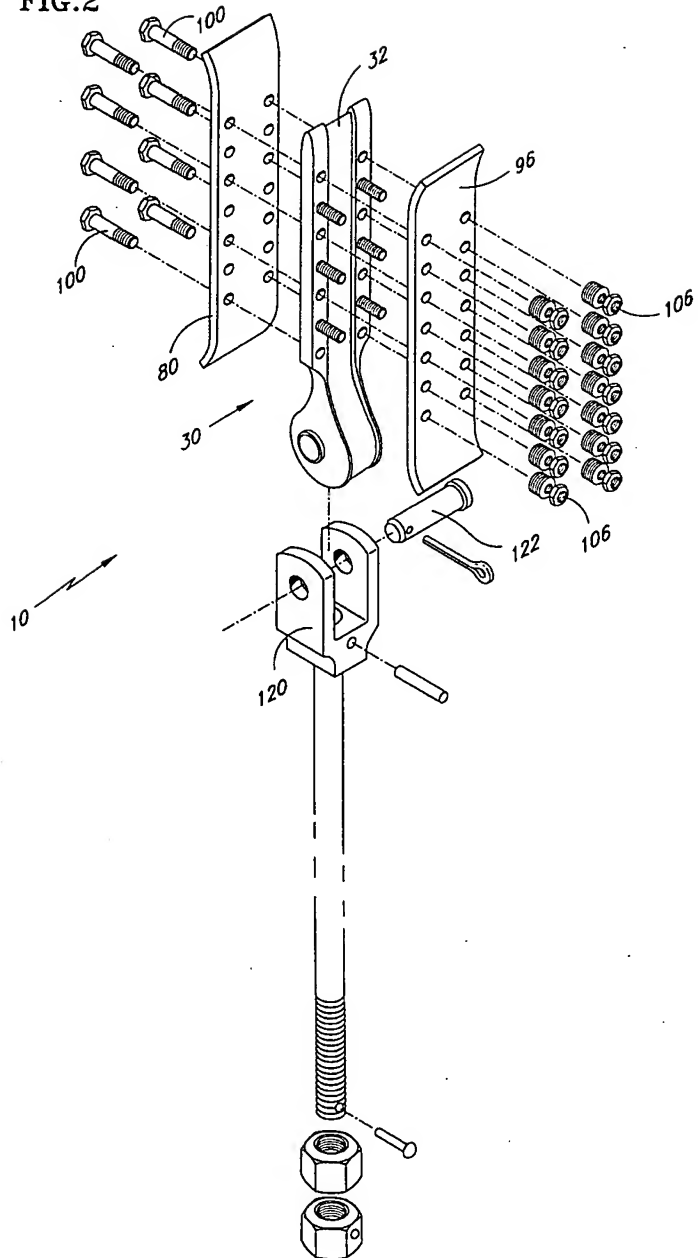


FIG. 2



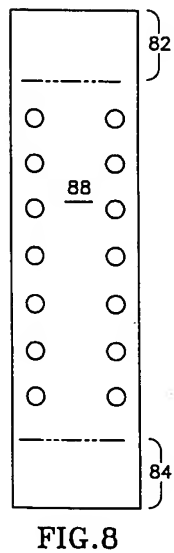
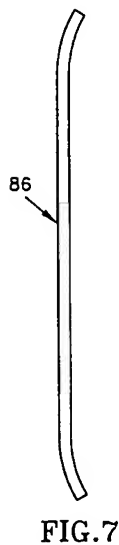
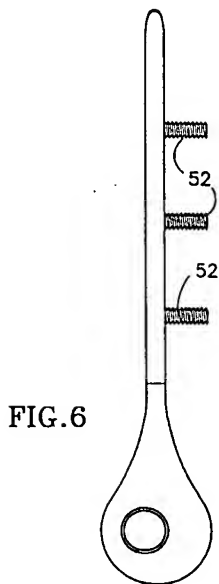
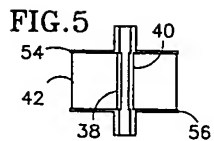
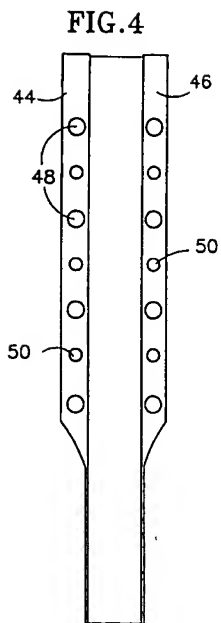
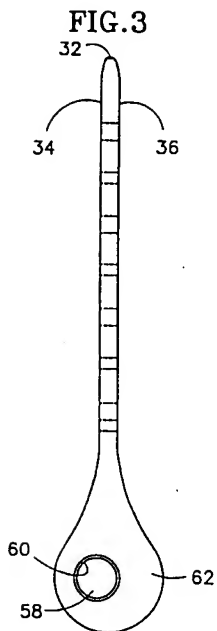


FIG.9

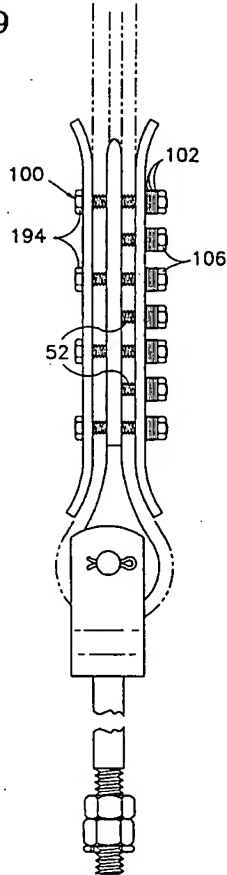


FIG.10

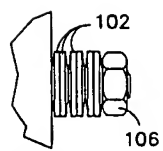
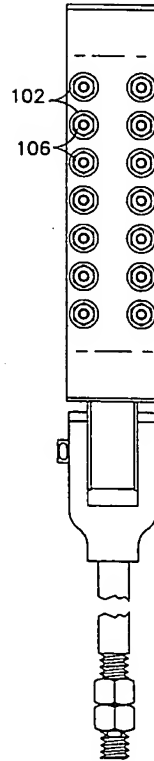


FIG.11

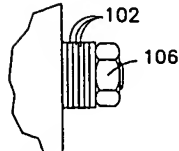


FIG.12

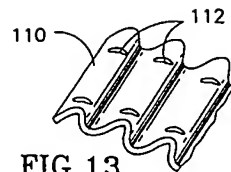


FIG.13

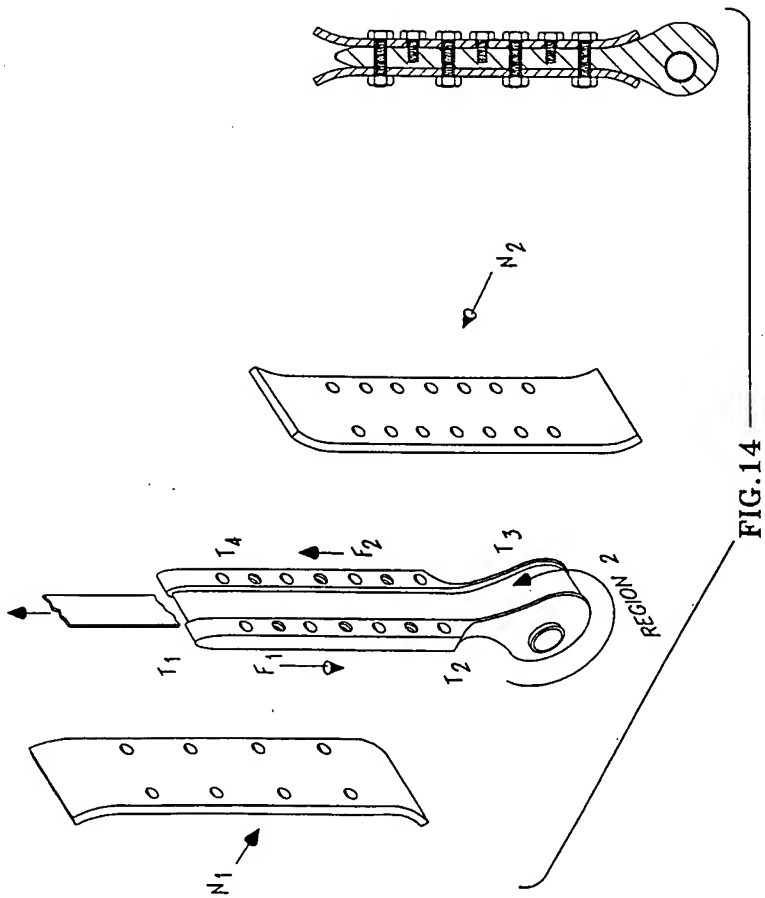
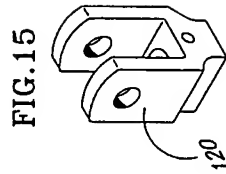


FIG. 16

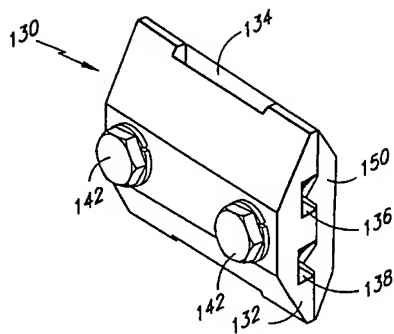


FIG. 17

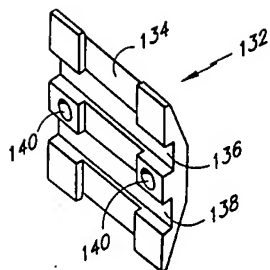
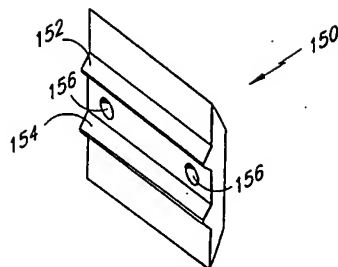


FIG. 18



INTERNATIONAL SEARCH REPORT

International Application No
PCT/US 99/03642

A. CLASSIFICATION OF SUBJECT MATTER

IPC 6 B66B7/08

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 6 B66B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X A	DD 115 089 A (BANYATERY BANYAZATI TERVEZO INTEZET) 12 September 1975 see claims; figures	15 1-3, 6, 11, 12, 16-19
A	FR 999 585 A (HEUER-HAMMER) 31 January 1952 see the whole document	1-21

☐ Further documents are listed in the continuation of box C.

☒ Patent family members are listed in annex.

* Special categories of cited documents:

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- *L* document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)
- *O* document referring to an oral disclosure, use, exhibition or other means
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- *T* later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
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Date of the actual completion of the international search

26 May 1999

Date of mailing of the international search report

01/06/1999

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Salvador, D

INTERNATIONAL SEARCH REPORT

information on patent family members

International Application No
PCT/US 99/03642

Patent document cited in search report		Publication date	Patent family member(s)	Publication date
DD 115089	A	12-09-1975	NONE	
FR 999585	A	31-01-1952	NONE	